



Driveline and Chassis Technology

# Design of wind turbine gearboxes with respect to noise

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Wind Power Technology

 **Hansen**<sup>®</sup>  
TRANSMISSIONS A brand of ZF




































# Agenda

- |    |   |
|----|---|
| 1. | <b>ZF Wind Power</b>                        |
| 2. | Mechanical noise of wind turbines           |
| 3. | Gearbox design for low noise and vibrations |
| 4. | Research project ALARM                      |
| 5. | Conclusions                                 |



# Combination of ZF and Hansen

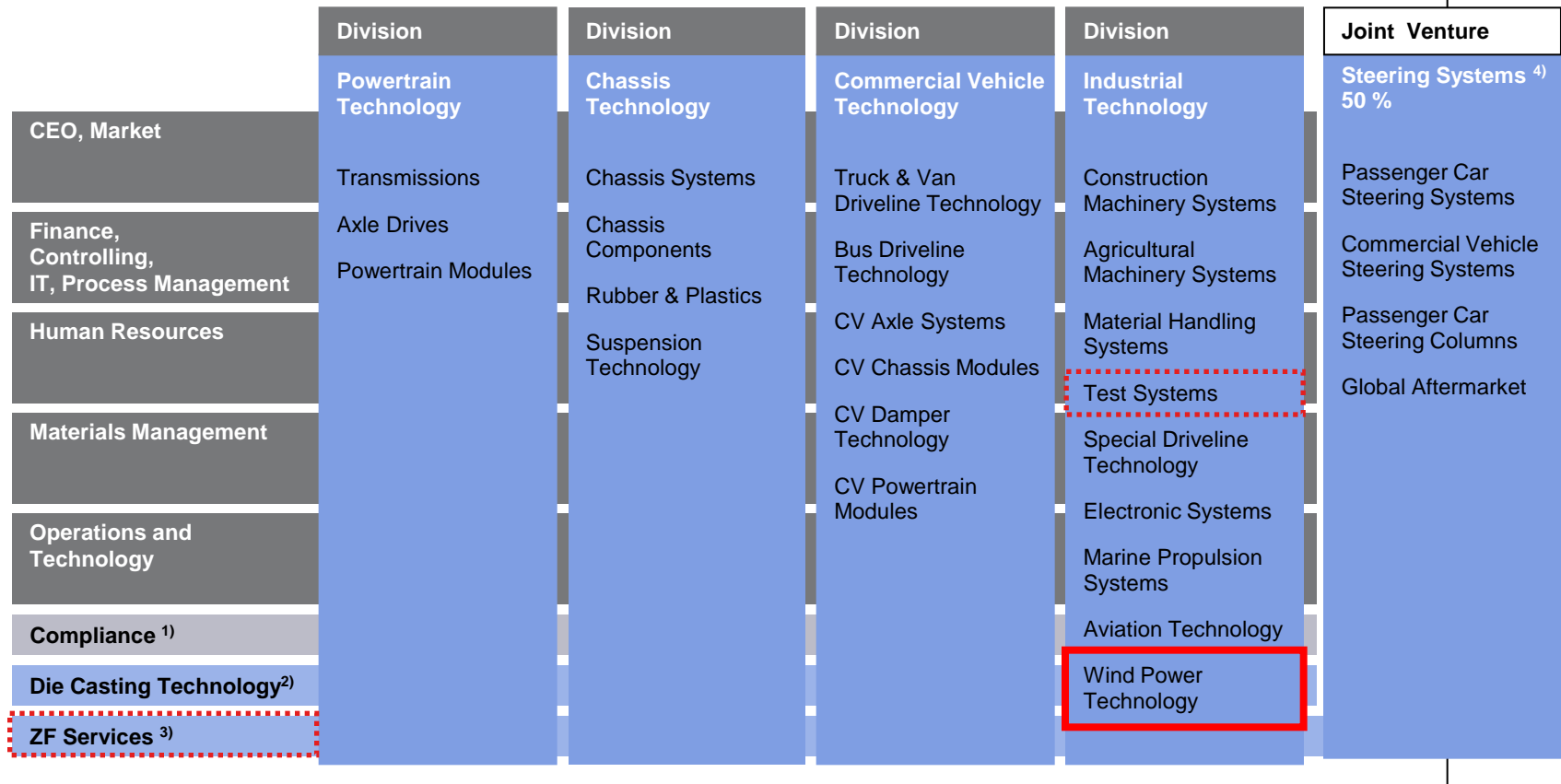
			 + 
<b>Customers</b>		     	     
<b>Products</b>	 2 MW	 1,5 MW to 6,15 MW	
<b>Regions</b>			
<b>Production Locations</b>	 Gainesville, USA	 Lommel, Belgium  Coimbatore, India  Tianjin, China	 Gainesville, USA  Lommel, Belgium  Coimbatore, India  Tianjin, China

Source: ZF; Hansen Transmission Annual Report 2011, 31.03.11 and Annual Report 2010, 31.03.10



# The Business Unit Wind Power Technology is part of the Industrial Technology Division of ZF

**ZF Friedrichshafen AG** Shareholders: 93.8 % Zeppelin Foundation and 6.2 % Dr. Jürgen and Irmgard Ulderup Foundation



Board of Management

Organizational unit

Integration of Hansen Transmissions

Partial activities in wind power

1) linked to CEO

2) linked to Materials Management domain

3) linked to Market domain

4) ZF Lenksysteme GmbH is a joint venture between ZF Friedrichshafen AG and Robert Bosch GmbH.



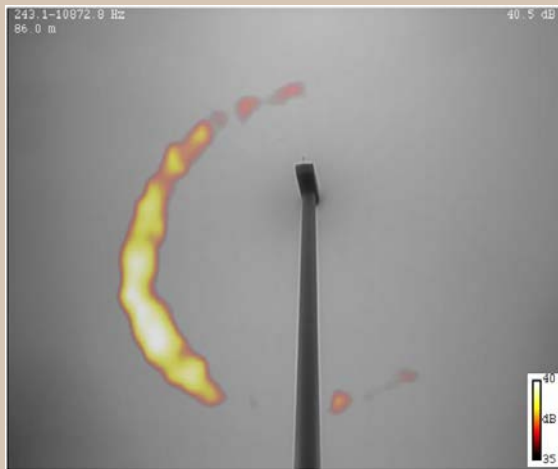
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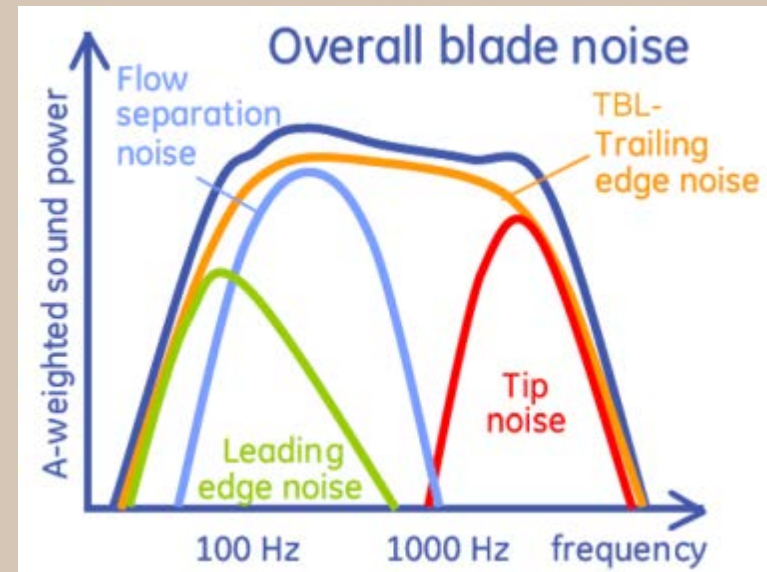
## Frequency spectrum of wind turbine noise sources

### 1. Aerodynamic noise

- source = blade motion in air
- primarily broadband - e.g. total SPL = 105 dB(A) (turbine at rated wind speed)



*Ramachandran et al. (2012) [2]*

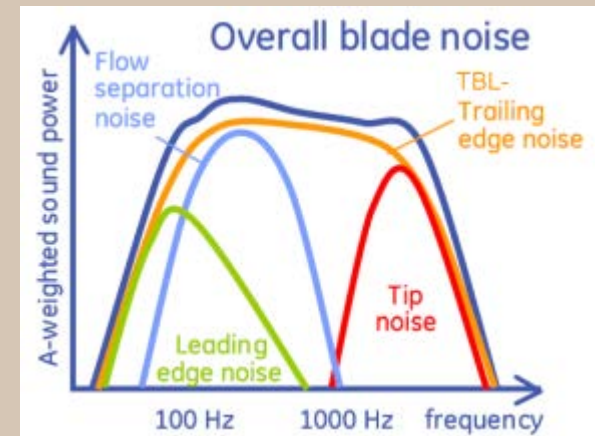
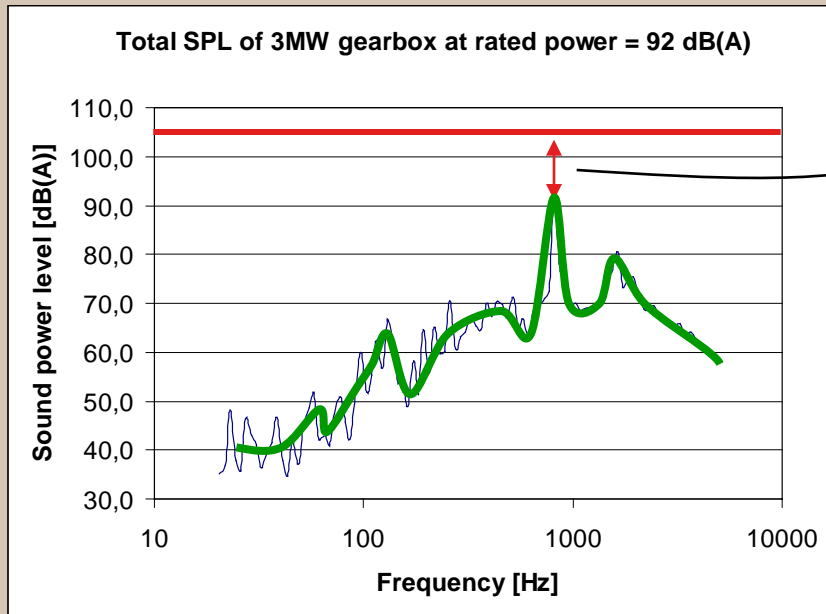


*Petitjean et al. (2011) [1]*

## Frequency spectrum of wind turbine noise sources

### 2. Mechanical noise

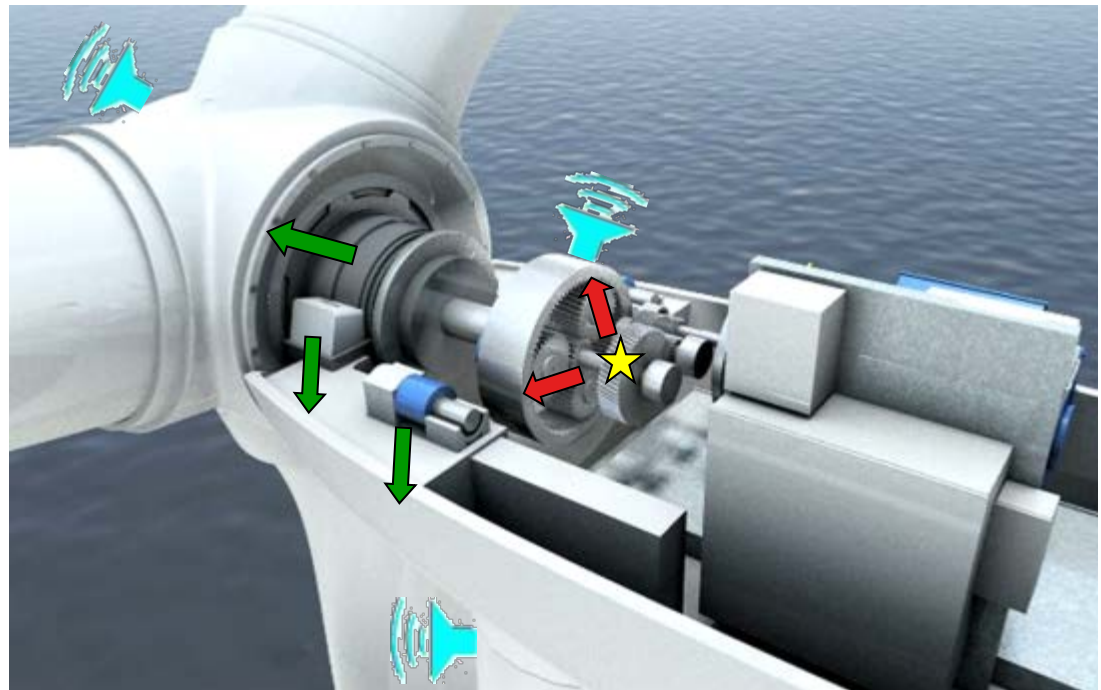
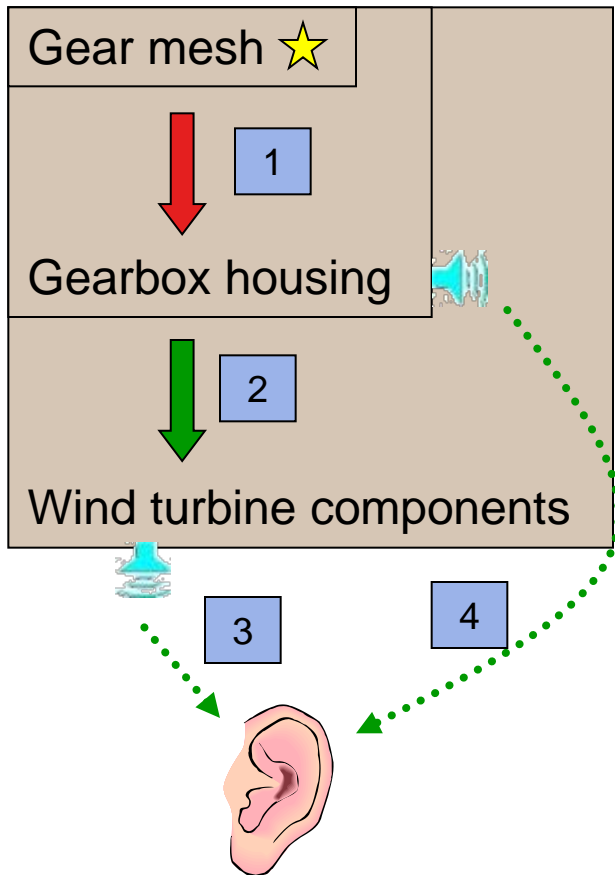
- sources: gearbox, generator, cooling fans, ...
- primarily tonal content, not determining WT overall SPL



*Petitjean et al. (2011) [1]*



## Transfer path of mechanical noise (gearbox as example)



Source: gear mesh, (bearings, pumps)



Structure borne transfer path **inside** / **outside** gearbox



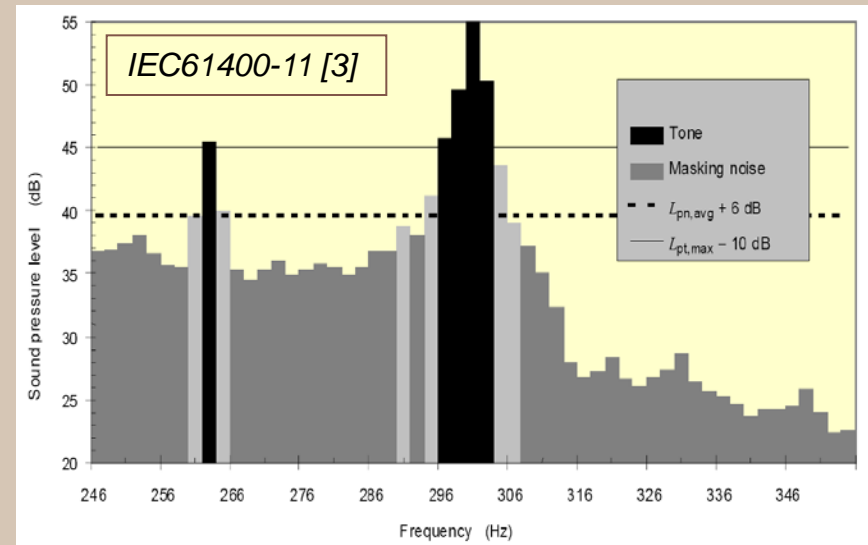
Airborne radiation by gearbox / by other components



## Potential annoyance

Annoyance depends on noise level of other sources (a.o. aerodynamic, background)

1. tonality:  
distinguishable tone is audible

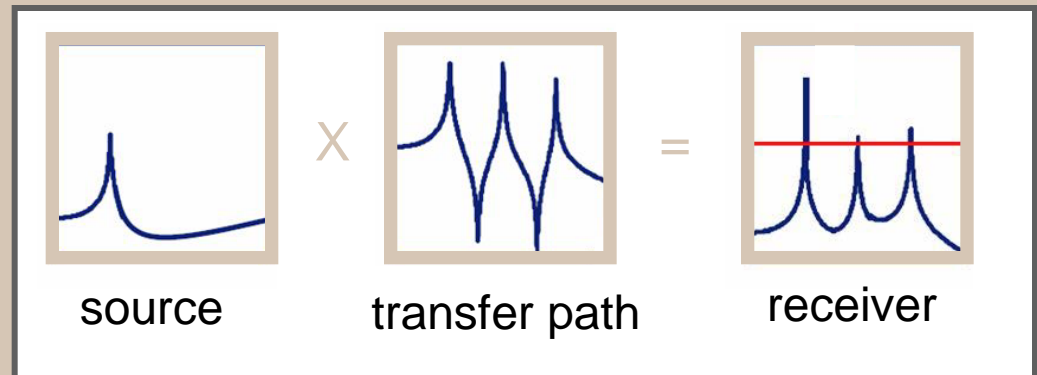


2. (overall WT sound power level dominated by mechanical noise)

## Measures / solutions for annoyance of mechanical noise

### A. Design “first time right”

- reduce source level
- optimise transfer path  
(avoid resonance / add damping)



### B. Corrective actions (when needed)


- shift resonance (e.g. tuned absorber, change speed)
- add more damping (e.g. nacelle insulation)

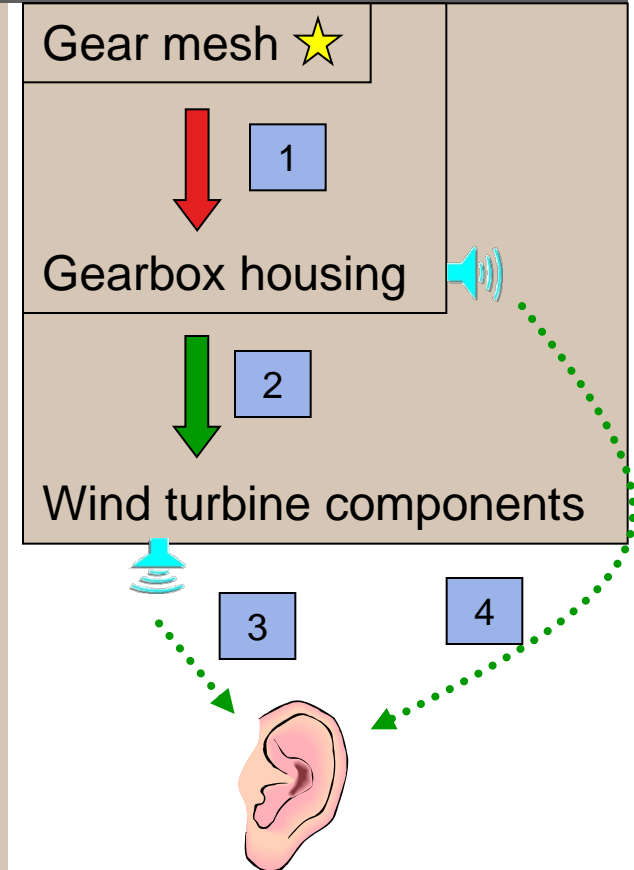
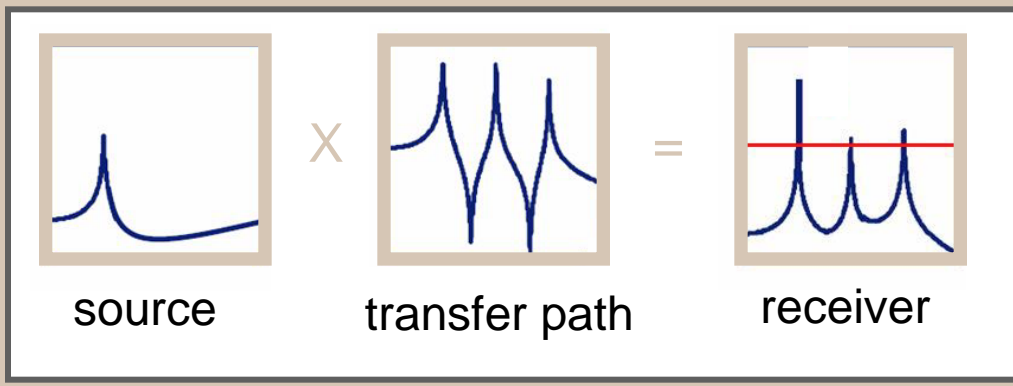


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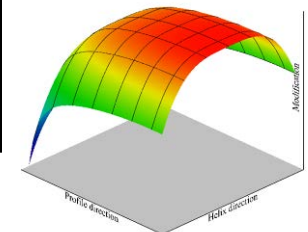
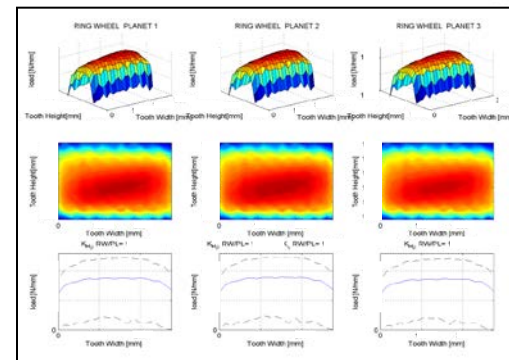
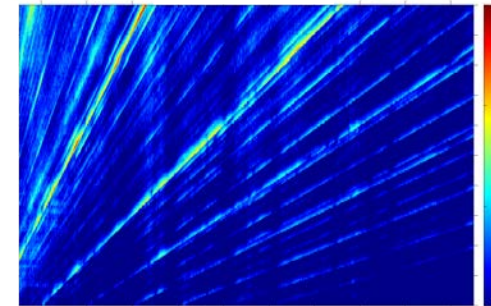
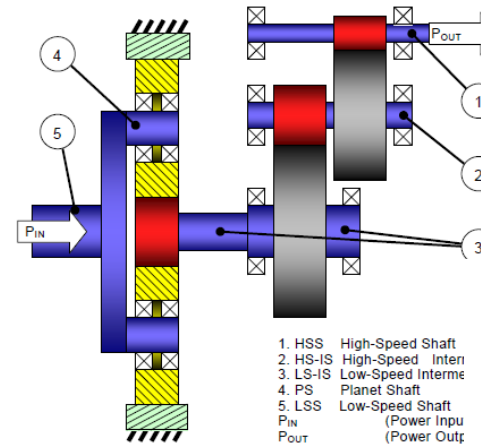
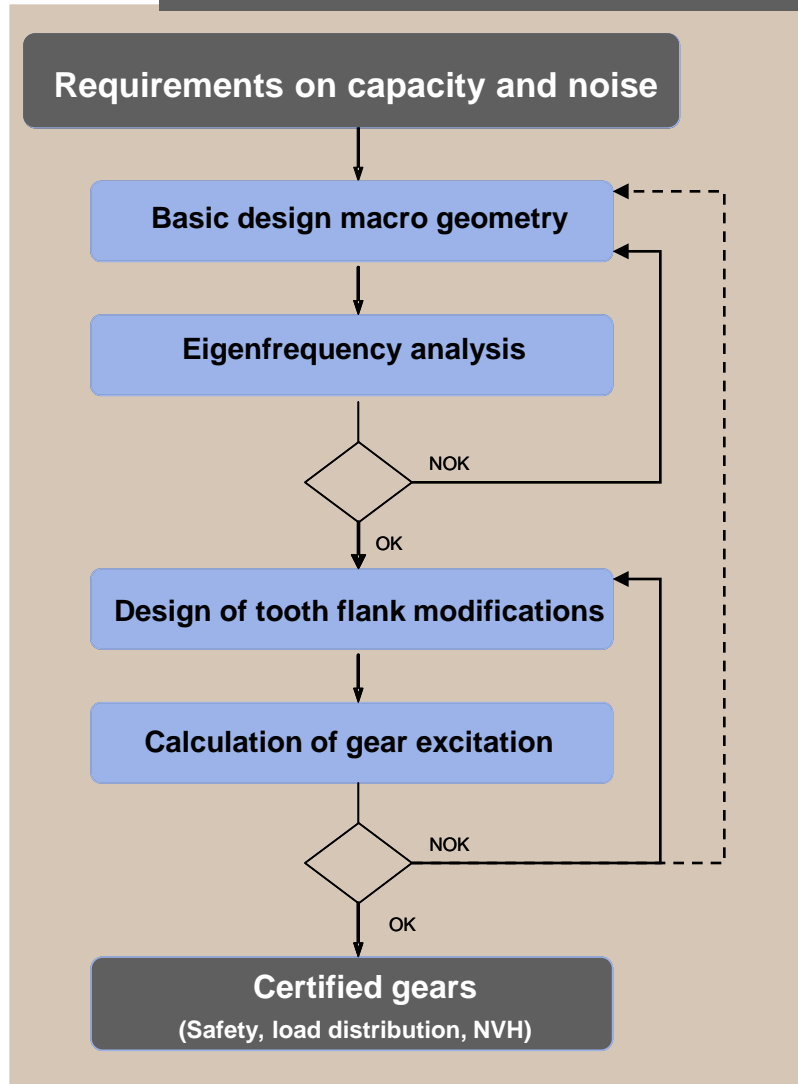
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## Challenges in “first time right” gearbox design

1. Reduce source level  
= reduce gear mesh excitation ★
2. Optimise gearbox housing
  - structure borne 1
  - airborne radiation 
3. Integration in wind turbine 2 3 4



## 1. Excitation reduction through gear optimisation

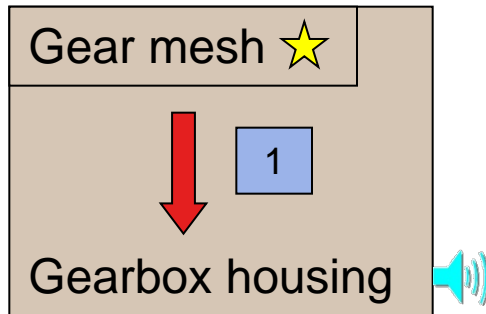




# Gearbox design for low noise and vibrations

## 2. Transfer path optimisation: structure borne and airborne

### Optimisation by virtual design assessment



#### Simulation requirements:

- Gear mesh model
- Transfer path structural model
- Radiation
  - structure borne at interfaces
  - airborne at housing

#### Chosen solution:

Multibody simulation for vibrations  
Acoustic FE simulation for airborne radiation

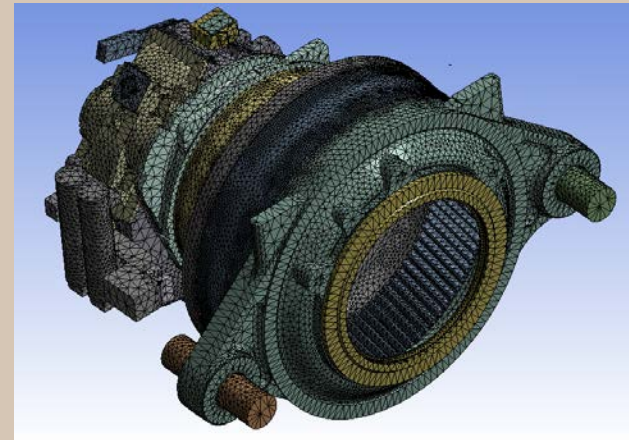
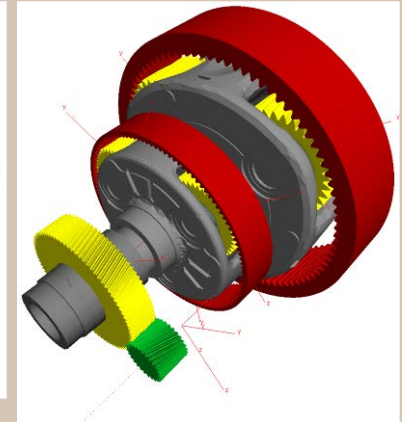
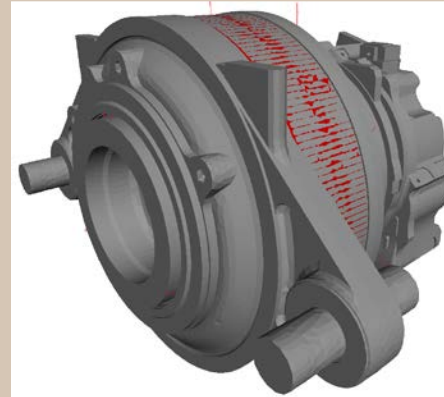


# Gearbox design for low noise and vibrations

## 2. Transfer path optimisation: structure borne and airborne

### Structural multibody simulation model in SIMPACK

- Gear excitation forces  
SIMPACK FE-225
- Bearing forces  
6x6 stiffness matrix
- Flexible components via CMS reduction:
  - planet carriers
  - shafts
  - housing: mesh size 5cm, 1st eigenmode < 100Hz, 1006 DOFs



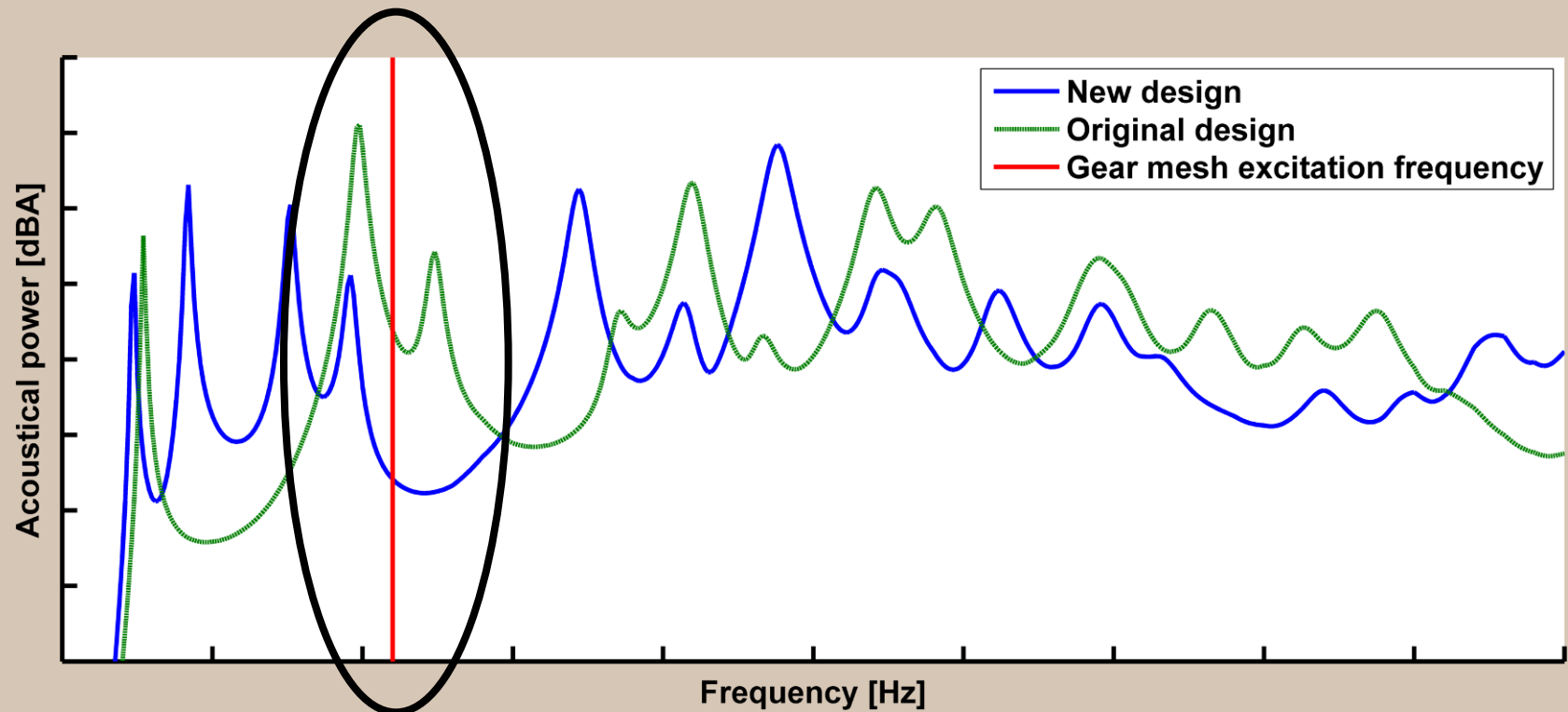




# Gearbox design for low noise and vibrations

## 2. Transfer path optimisation: structure borne and airborne

### Example of a housing design optimisation to avoid resonance





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# Research project ALARM

**“First time right” wind turbine design requires cooperation between stakeholders**

Cooperation between 3 stakeholders in source and transfer path:

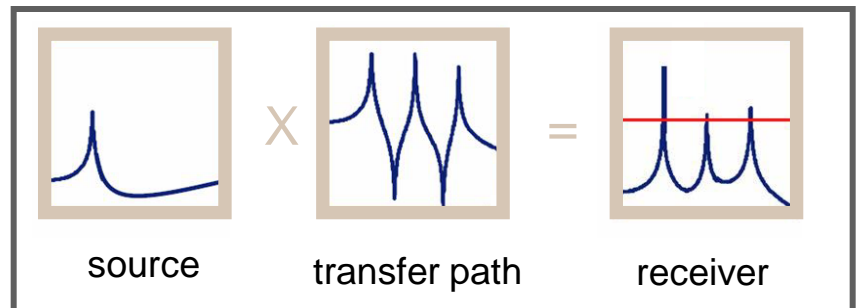
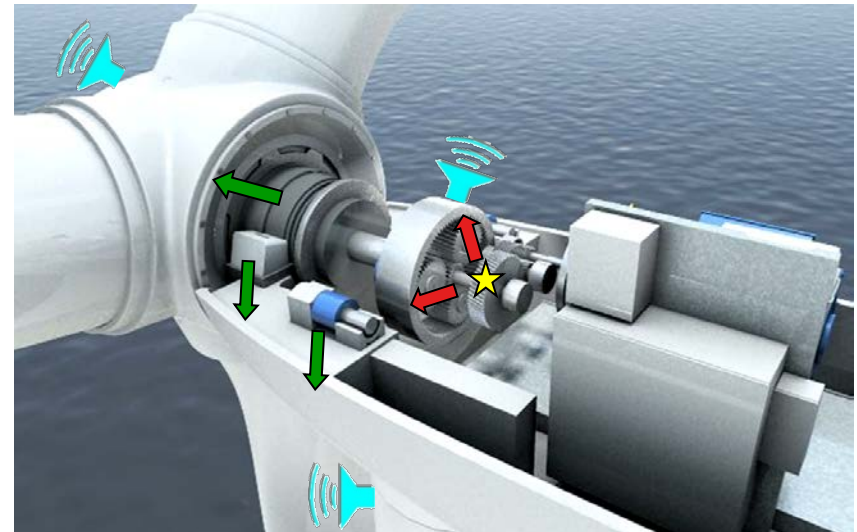
	WT manufacturer	Hamburg
	WT gearbox manufacturer	Antwerp
	Bushing manufacturer	Rimbach

+ 2 engineering companies

	Software / Simulation	Hamburg
	Software & Hardware / Simulation & Test	Leuven

+ 2 universities

	Chair of Eng. Design	Erlangen
	Dept. Mechanical Eng.	Leuven

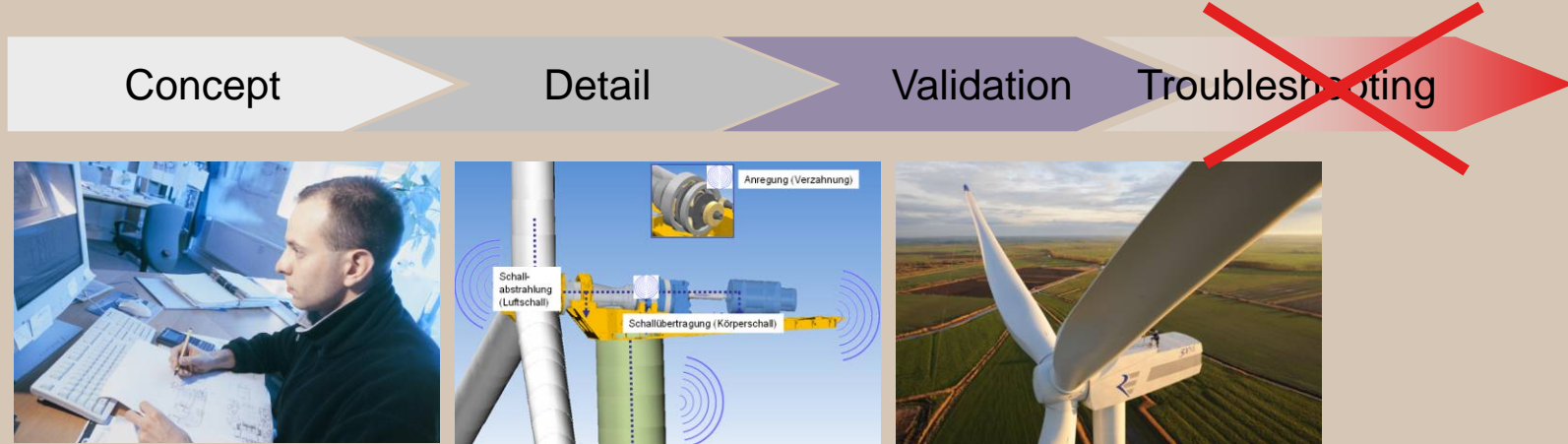




# Research project ALARM

## „Assistenzsystem zur Lärmreduzierten Auslegung Rotierender Maschinen“

Goal: Save money and time in the development of wind turbines, gearboxes and elastomer bushings by avoiding tonal noise issues in the field



Total budget:

German partners: 2.0 M€

Belgian partners: 1.8 M€

with governmental support  
~ high level of innovation

Duration: 3 years (01.03.2012 – 28.02.2015)



# Research project ALARM

## ALARM research activities

### Simulations & Measurements

#### Data acquisition

Test rig measurements  
→ Gear mesh excitation

WT measurement (a.o. TPA)  
→ structure and airborne radiation

Measurements: WT, high freq shaker  
→ uncoupling structure borne transfer

Simulation WT  
→ structure and airborne radiation

Simulation test rig  
→ Gear mesh excitation

Measurements test rig, WT  
→ Gear modelling, TPA



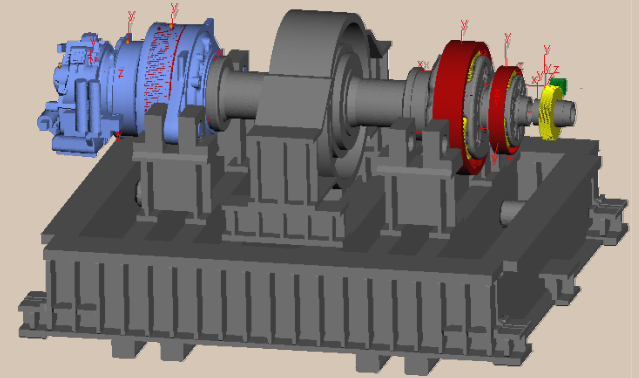
# Research project ALARM

## Repower 3.2M114 wind turbine

### Status

#### ZF gearbox

- gearbox simulation models ready
- 1st proto gearbox test rig measurements done
- gearbox assembly in wind turbine planned



### Outlook (planned for 2013)

- additional test rig measurements
- elaborate field measurement campaign
- correlation of simulations with experimental results





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# Conclusions

## 1. Mechanical noise of wind turbines

- ⇒ lower in overall sound power level compared to aerodynamic noise
- ⇒ typically tonal and mainly structure borne noise
- ⇒ avoid annoyance (= tonality) by reducing source level and optimising transfer path

## 2. Gearbox design for low noise and vibrations

- ⇒ excitation reduction by optimising macro and microgeometry
- ⇒ transfer path optimisation by using MBS and acoustic FE to predict the response
- ⇒ additional analysis at wind turbine level enhances optimisation potential at gearbox level

## 3. Research project ALARM

- ⇒ 7 industrial partners cooperating on 'expert design system' to avoid tonality issues in the field
- ⇒ focus on Repower 3.2M114 wind turbine development:
  - ⇒ simulation of noise sources, transfer paths and radiation
  - ⇒ experimental analyses on test rigs and in the field to build up know how and verify models



Thank you very much for your attention!



# Measuring wind turbine gearboxes

## Suggestion for an efficient measurement strategy for WT gearboxes

1. Wind turbine should comply with (local) acoustic regulations

Measurements in prototype phase



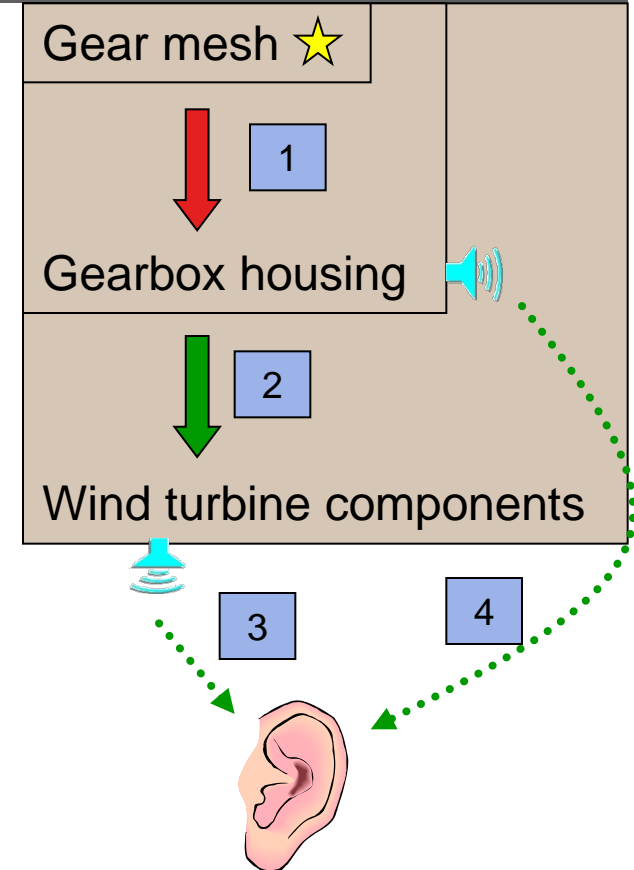
Correlation between test rig and in-the-field

2. Gearbox should meet general quality requirements

Test rig measurements in serial production



Identify quality deviations





# References

- [1] Dr. Benoit Petitjean, Dr. Roger Drobietz, Dr. Kevin Kinzie, **Wind Turbine Blade Noise Mitigation Technologies**, in *Fourth International Meeting on Wind Turbine Noise, Rome Italy 12-14 April 2011*
- [2] Rakesh C. Ramachandran, Hirenkumar Patel and Ganesh Raman, **Localization of wind turbine noise sources using a compact microphone array with advanced beamforming algorithms**, in Berlin Beamforming Conference 2012
- [3] IEC61400-11 Ed.3: **Wind turbines - Part 11: Acoustic noise measurement techniques**



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